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## Navier-Stokes Equations

The Navier-Stokes equations are the fundamental partial differential equations that describe the flow of **incompressible fluids**. Using the rate of **stress** and rate of **strain** tensors, it can be shown that the components  $F_j$  of a **viscous force**  $\mathbf{F}$  in a nonrotating frame are given by

$$\frac{F_i}{V} = \frac{\partial}{\partial x_j} \left[ \eta \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) + \lambda \delta_{ij} \nabla \cdot \mathbf{u} \right] \quad (1)$$

$$= \frac{\partial}{\partial x_j} \left[ \eta \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} - \frac{2}{3} \delta_{ij} \nabla \cdot \mathbf{u} \right) + \mu_B \delta_{ij} \nabla \cdot \mathbf{u} \right], \quad (2)$$

(Tritton 1988, Faber 1995), where  $\eta$  is the **dynamic viscosity**,  $\lambda$  is the **second viscosity coefficient**,  $\delta_{ij}$  is the **Kronecker delta**,  $\nabla \cdot \mathbf{u}$  is the **divergence**,  $\mu_B$  is the **bulk viscosity**, and **Einstein summation** has been used to sum over  $j = 1, 2,$  and  $3$ .

Now, for an **incompressible fluid**, the **divergence**  $\nabla \cdot \mathbf{u} = 0$ , so the  $\lambda$  term drops out. Taking  $\eta$  to be constant in space and writing the remainder of (2) in vector form then gives

$$\frac{\mathbf{F}_{\text{viscous}}}{V} = \eta \nabla^2 \mathbf{u}, \quad (3)$$

where  $\nabla^2 \mathbf{u}$  is the **vector Laplacian**.

There are two additional forces acting on fluid parcels, namely the **pressure force**

$$\frac{\mathbf{F}_{\text{pressure}}}{V} = -\nabla P, \quad (4)$$

where  $P$  is the pressure, and the so-called body force

$$\mathcal{F} = \frac{\mathbf{F}_{\text{body}}}{V}. \quad (5)$$

Adding the three forces (3), (4), and (5) and equating them to **Newton's law for fluids** yields the equation

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla P + \eta \nabla^2 \mathbf{u} + \mathcal{F}, \quad (6)$$

and dividing through by the **density**  $\rho$  gives

$$\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} = -\frac{\nabla P}{\rho} + \nu \nabla^2 \mathbf{u} + \frac{\mathcal{F}}{\rho}, \quad (7)$$

where the **kinematic viscosity** is defined by

$$\nu \equiv \frac{\eta}{\rho}. \quad (8)$$

The vector equations (7) are the (irrotational) Navier-Stokes equations.

When combined with the [continuity equation](#) of fluid flow, the Navier-Stokes equations yield four equations in four unknowns (namely the scalar  $\rho$  and vector  $\mathbf{u}$ ). However, except in degenerate cases in very simple geometries (such as [Stokes flow](#), these equations cannot be solved exactly, so approximations are commonly made to allow the equations to be solved approximately.

As it must, the Navier-Stokes equations satisfy [conservation of mass](#), [momentum](#), and [energy](#). [Mass conservation](#) is included implicitly through the [continuity equation](#),

$$\frac{\partial \rho}{\partial t} = -\nabla \cdot \Phi_{\text{mass}} = -\nabla \cdot (\rho \mathbf{u}) = -\mathbf{u} \cdot \nabla \rho - \rho \nabla \cdot \mathbf{u}. \quad (9)$$

So, for an [incompressible fluid](#),

$$\frac{\partial \rho}{\partial t} = -\rho \nabla \cdot \mathbf{u}. \quad (10)$$

[Conservation of momentum](#) requires

$$\frac{\text{momentum}}{\text{mass}} = [\text{mass}] + [\text{pressure}] + [\text{body force}] + [\text{viscosity}], \quad (11)$$

so

$$\frac{\partial \mathbf{u}}{\partial t} = (\mathbf{u} \cdot \nabla) \mathbf{u} - \frac{1}{\rho} \nabla P + \frac{\mathcal{F}}{\rho} + \nu \nabla^2 \mathbf{u}. \quad (12)$$

Finally, [conservation of energy](#) follows from

$$\frac{\partial s}{\partial t} = -\mathbf{u} \cdot \nabla s + \frac{Q}{T}, \quad (13)$$

where  $s$  is the [entropy](#) per unit mass,  $Q$  is the heat transferred, and  $T$  is the [temperature](#).

Now consider the irrotational Navier-Stokes equations in particular coordinate systems. In [Cartesian coordinates](#) with the components of the velocity vector given by  $\mathbf{u} = (u, v, w)$ , the [continuity equation](#) is

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (14)$$

and the Navier-Stokes equations are given by

$$\rho \left( \frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = -\frac{\partial P}{\partial x} + \eta \left( \frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) + F_x \quad (15)$$

$$\rho \left( \frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right) = -\frac{\partial P}{\partial y} + \eta \left( \frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right) + F_y \quad (16)$$

$$\rho \left( \frac{\partial w}{\partial t} + u \frac{\partial w}{\partial x} + v \frac{\partial w}{\partial y} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial P}{\partial z} + \eta \left( \frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} + \frac{\partial^2 w}{\partial z^2} \right) + F_z. \quad (17)$$

In [cylindrical coordinates](#) with the components of the velocity vector given by  $\mathbf{u} = (u_r, u_\theta, u_z)$ , the [continuity equation](#) is

$$\frac{\partial u_r}{\partial r} + \frac{u_r}{r} + \frac{1}{r} \frac{\partial u_\phi}{\partial \phi} + \frac{\partial u_z}{\partial z} = 0, \quad (18)$$

and the Navier-Stokes equations are given by

$$\begin{aligned} & \rho \left( \frac{\partial u_r}{\partial t} + u_r \frac{\partial u_r}{\partial r} + \frac{u_\phi}{r} \frac{\partial u_r}{\partial \phi} + u_z \frac{\partial u_r}{\partial z} - \frac{u_\phi^2}{r} \right) \\ &= -\frac{\partial P}{\partial r} + \eta \left( \frac{\partial^2 u_r}{\partial r^2} + \frac{1}{r} \frac{\partial u_r}{\partial r} - \frac{u_r}{r^2} + \frac{1}{r^2} \frac{\partial^2 u_r}{\partial \phi^2} + \frac{\partial^2 u_r}{\partial z^2} - \frac{2}{r^2} \frac{\partial u_\phi}{\partial \phi} \right) + F_r \end{aligned} \quad (19)$$

$$\begin{aligned} & \rho \left( \frac{\partial u_\phi}{\partial t} + u_r \frac{\partial u_\phi}{\partial r} + \frac{u_r u_\phi}{r} + \frac{u_\phi}{r} \frac{\partial u_\phi}{\partial \phi} + u_z \frac{\partial u_\phi}{\partial z} \right) \\ &= -\frac{1}{r} \frac{\partial P}{\partial \phi} + \eta \left( \frac{\partial^2 u_\phi}{\partial r^2} + \frac{1}{r} \frac{\partial u_\phi}{\partial r} - \frac{u_\phi}{r^2} + \frac{1}{r^2} \frac{\partial^2 u_\phi}{\partial \phi^2} + \frac{\partial^2 u_\phi}{\partial z^2} + \frac{2}{r^2} \frac{\partial u_r}{\partial \phi} \right) + F_\phi \end{aligned} \quad (20)$$

$$\begin{aligned} & \rho \left( \frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\phi}{r} \frac{\partial u_z}{\partial \phi} + u_z \frac{\partial u_z}{\partial z} \right) \\ &= -\frac{\partial P}{\partial z} + \eta \left( \frac{\partial^2 u_z}{\partial r^2} + \frac{1}{r} \frac{\partial u_z}{\partial r} + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \phi^2} + \frac{\partial^2 u_z}{\partial z^2} \right) + F_z. \end{aligned} \quad (21)$$

In spherical coordinates [②](#) with the components of the velocity vector given by  $\mathbf{u} = (u_r, u_\theta, u_\phi)$ , the continuity equation is

$$\frac{\partial u_r}{\partial r} + \frac{2u_r}{r} + \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_\theta \cot \theta}{r} + \frac{1}{r \sin \theta} \frac{\partial u_\phi}{\partial \phi} = 0 \quad (22)$$

and the Navier-Stokes equations are given by

$$\begin{aligned} & \rho \left( \frac{\partial u_r}{\partial t} + u_r \frac{\partial u_r}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_r}{\partial \theta} + \frac{u_\phi}{r \sin \theta} \frac{\partial u_r}{\partial \phi} - \frac{u_\theta^2}{r} - \frac{u_\phi^2}{r} \right) \\ &= -\frac{\partial P}{\partial r} + \eta \left( \frac{\partial^2 u_r}{\partial r^2} + \frac{2}{r} \frac{\partial u_r}{\partial r} - \frac{2u_r}{r^2} + \frac{1}{r^2} \frac{\partial^2 u_r}{\partial \theta^2} + \frac{\cot \theta}{r^2} \frac{\partial u_r}{\partial \theta} \right. \\ & \left. + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 u_r}{\partial \phi^2} - \frac{2}{r^2} \frac{\partial u_\theta}{\partial \theta} - \frac{2u_\theta \cot \theta}{r^2} - \frac{2}{r^2 \sin \theta} \frac{\partial u_\phi}{\partial \phi} \right) + F_r \end{aligned} \quad (23)$$

$$\begin{aligned} & \rho \left( \frac{\partial u_\theta}{\partial t} + u_r \frac{\partial u_\theta}{\partial r} + \frac{u_r u_\theta}{r} + \frac{u_\theta}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_\phi}{r \sin \theta} \frac{\partial u_\theta}{\partial \phi} - \frac{u_\phi^2 \cot \theta}{r} \right) \\ &= -\frac{1}{r} \frac{\partial P}{\partial \theta} + \eta \left( \frac{\partial^2 u_\theta}{\partial r^2} + \frac{2}{r} \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r^2 \sin^2 \theta} + \frac{1}{r^2} \frac{\partial^2 u_\theta}{\partial \theta^2} + \frac{\cot \theta}{r^2} \frac{\partial u_\theta}{\partial \theta} + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 u_\theta}{\partial \phi^2} + \frac{2}{r^2} \frac{\partial u_r}{\partial \theta} - \frac{2 \cot \theta}{r^2 \sin \theta} \frac{\partial u_\phi}{\partial \phi} \right) + F_\theta \end{aligned} \quad (24)$$

$$\begin{aligned} & \rho \left( \frac{\partial u_\phi}{\partial t} + u_r \frac{\partial u_\phi}{\partial r} + \frac{u_r u_\phi}{r} + \frac{u_\theta}{r} \frac{\partial u_\phi}{\partial \theta} + \frac{u_\phi u_\theta \cot \theta}{r} + \frac{u_\phi}{r \sin \theta} \frac{\partial u_\phi}{\partial \phi} \right) \\ &= -\frac{1}{r \sin \theta} \frac{\partial P}{\partial \phi} + \eta \left( \frac{\partial^2 u_\phi}{\partial r^2} + \frac{2}{r} \frac{\partial u_\phi}{\partial r} - \frac{u_\phi}{r^2 \sin^2 \theta} + \frac{1}{r^2} \frac{\partial^2 u_\phi}{\partial \theta^2} + \frac{\cot \theta}{r^2} \frac{\partial u_\phi}{\partial \theta} + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 u_\phi}{\partial \phi^2} + \frac{2}{r^2 \sin \theta} \frac{\partial u_r}{\partial \phi} + \frac{2 \cot \theta}{r^2 \sin \theta} \frac{\partial u_\theta}{\partial \phi} \right) + F_\phi. \end{aligned} \quad (25)$$

The Navier-Stokes equations with no body force (i.e.,  $\mathcal{F} = \mathbf{0}$ )

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla P + \eta \nabla^2 \mathbf{u} \quad (26)$$

can be put into dimensionless form using the definitions

$$x' \equiv \frac{x}{L} \quad (27)$$

$$y' \equiv \frac{y}{L} \quad (28)$$

$$z' \equiv \frac{z}{L} \quad (29)$$

$$\mathbf{u}' \equiv \frac{\mathbf{u}}{U} \quad (30)$$

$$.. \quad P$$

$$P' = \frac{-}{\rho U^2} \quad (31)$$

$$\nabla' \equiv \hat{x} \frac{\partial}{\partial x'} + \hat{y} \frac{\partial}{\partial y'} + \hat{z} \frac{\partial}{\partial z'} = L \nabla \quad (32)$$

$$t' \equiv t \frac{U}{L} \quad (33)$$

Here,  $U$  and  $L$  are a characteristic velocity and a characteristic length. Then

$$\rho \frac{\partial(U\mathbf{u}')}{\partial \left(\frac{L}{U}t'\right)} + \rho U \mathbf{u}' \cdot \frac{1}{L} \nabla' U \mathbf{u}' = -\frac{1}{L} \nabla' (\rho U^2 P') + \eta \frac{1}{L^2} \nabla'^2 (U \mathbf{u}'). \quad (34)$$

Assuming constant  $\rho$  and multiplying both sides by  $L/(\rho U^2)$  gives

$$\frac{\partial \mathbf{u}'}{\partial t'} + \mathbf{u}' \cdot \nabla' \mathbf{u}' = -\nabla' P' + \frac{\eta}{L \rho U} \nabla'^2 \mathbf{u}' \quad (35)$$

$$= -\nabla' P' + \frac{1}{\text{Re}} \nabla'^2 \mathbf{u}', \quad (36)$$

where  $\text{Re}$  is a dimensionless parameter known as the [Reynolds number](#). Pressure is a parameter fixed by the observer, so it follows that the only other force is inertia force. Furthermore, the relative magnitudes of the pressure and inertial forces are describe by the [Reynolds number](#), defined as

$$\text{Re} = \frac{F_{\text{inertia}}}{F_{\text{viscous}}}. \quad (37)$$

For irrotational, incompressible flow with  $\mathbf{F} = \mathbf{0}$ , the Navier-Stokes equation then simplifies to

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla P + \eta \nabla^2 \mathbf{u}. \quad (38)$$

For low [Reynolds number](#), the inertia term is smaller than the viscous term and can therefore be ignored, leaving the equation of creeping motion

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \nabla P = \eta \nabla^2 \mathbf{u}. \quad (39)$$

In this regime, viscous interactions have an influence over large distances from an obstacle. For low [Reynolds number](#) flow at low [pressure](#), the Navier-Stokes equation becomes a diffusion equation

$$\rho \frac{\partial \mathbf{u}}{\partial t} = \eta \nabla^2 \mathbf{u}. \quad (40)$$

For high [Reynolds number](#) flow, the viscous force is small compared to the inertia force, so it can be neglected, leaving [Euler's equation of inviscid motion](#)

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla P. \quad (41)$$

In the absence of a [pressure](#) force,

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho \mathbf{u} \cdot \nabla \mathbf{u} = \eta \nabla^2 \mathbf{u}, \quad (42)$$

which can be written as

$$\nabla \times \frac{\partial \mathbf{u}}{\partial t} - \nabla \times [\mathbf{u} \times (\nabla \times \mathbf{u})] = \nu \nabla^2 (\nabla \times \mathbf{u}). \quad (43)$$

For steady incompressible flow,

$$\frac{\partial \mathbf{u}}{\partial t} = \mathbf{0}. \quad (44)$$

At low Reynolds number,

$$\nabla P = \eta \nabla^2 \mathbf{u}. \quad (45)$$

At low Reynolds number and low pressure

$$\nabla^2 \mathbf{u} = \mathbf{0}. \quad (46)$$

At high Reynolds number

$$\rho \mathbf{u} \cdot \nabla \mathbf{u} = -\nabla P. \quad (47)$$

For small pressure forces,

$$\rho \mathbf{u} \cdot \nabla \mathbf{u} = \eta \nabla^2 \mathbf{u}, \quad (48)$$

which can be written as

$$-\nabla \times [\mathbf{u} \times (\nabla \times \mathbf{u})] = \nu \nabla^2 (\nabla \times \mathbf{u}). \quad (49)$$

**SEE ALSO:** [Continuity Equation](#), [Euler's Equation of Inviscid Motion](#), [Navier-Stokes Equations--Rotational](#), [Reynolds Number](#), [Stokes Flow](#), [Stokes Flow--Cylinder](#), [Stokes Flow--Sphere](#)

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